

**Amendments to the Specification and Abstract:**

**Please replace paragraph [0013] on page 5 of the specification with the following new paragraph [0013].**

[0013] Further, in mounting the tapered roller bearing on a mounting portion, if the gap between the large end faces 59 of the tapered rollers 57 and the large rib surface 54 is large in the initial assembled state shown in Fig. 12A, break-in time tends to be long until the tapered rollers 57 settle in their regular positions shown in Fig. 12C. As shown in Fig. 11, since the small rib surface 55 of the inner ring 56 is formed inclined outwardly relative to the ~~the~~ small end faces 60 of the tapered rollers 57, variation in the gap between the large end faces 59 and the large rib surface 54 in the initial assembled state is large for the following reasons, and the abovementioned break-in time until all the tapered rollers 57 settle in their regular positions tends to become even longer.

**Please replace paragraph [0018] spanning pages 5 and 6 of the specification with the following new paragraph [0018].**

[0018] According to this invention, there is provided a tapered roller bearing comprising an outer ring having a conical raceway, an inner ring having a conical raceway and formed with a large rib surface on the large diameter side of the conical raceway, a plurality of tapered rollers rollably arranged between the raceway of the outer ring and the raceway of the inner ring, and a retainer for keeping the tapered rollers circumferentially spaced a predetermined distance from each other, ~~characterized in that the~~ other. The outer ring, the inner ring and the tapered rollers are all formed from a steel having an oxygen content of 9 ppm or less, and ~~that a~~ carbo-nitrided layer having a carbon content of 0.80 wt% or more and a Rockwell hardness HRC of 58 or more is formed on surfaces of the outer ring, the inner ring and the tapered rollers, and ~~that the~~ retained austenite content of the carbo-nitrided layer is 25 to 35 vol%.

**Please replace paragraph [0060] spanning pages 15 and 16 of the specification with the following new paragraph [0060].**

**[0060]** With reference to Figs. 1 and ~~through~~ 9, embodiments of this invention are described. Fig. 1 shows, as described above, a differential of an automobile, in which for the support of the drive pinion 4 and the differential gear case 7 on which is mounted the ring gear 5, the gear shaft support device using the tapered roller bearings 2, 3, 6 of the embodiments is adopted.

**Please replace paragraph [0098] on page 28 of the specification with the following new paragraph [0098].**

**[0098]** Fig. 9 shows a portion of the tapered roller bearing of the fifth embodiment. This tapered roller bearing was also used for the support of a differential gear case 7 as shown in Fig. 1. The large rib surface 41 of the inner ring 40 comprises a conical surface 41a, and a flank 41b smoothly connecting with the conical surface 41a and having an arcuate section, and a chamfer 41c connecting with the flank 41b. The conical surface 41a is, like the tapered roller bearing shown in Fig. 5, formed with point O as its center. The end faces 43 of the tapered rollers 42 are each formed as a spherical surface 43a having a radius of curvature R that is smaller than the distance Ro from point O to the large rib surface 41 of the inner ring 40. A recess 44 of a circular shape is formed at the center of the spherical surface 43a. The outer peripheral end of the recess 44 extends to near the boundary between the conical surface 41a and the flank 41b of the large rib surface 41.